Development of a Flow Characterization Sensor for Two-Dimensional Fluid Flows

Gerald Wedel, Thomas Wiesner, Thomas Egeln

March 24th, 2025



HELMHOLTZ ZENTRUM DRESDEN ROSSENDORF

Member of the Helmholtz Association

Topics

1 Problem definition

- 2 Mechanical constructions
- 3 The Pressure Sensors
- 4 Frequency Response
- 5 Media Compatibility Tests
- 6 Tubing Material Tests
- 7 Measurements/data evaluation





Member of the Helmholtz Association

Flow characterization approach

- Characterisation of the flow between two adjacent plates
- Sensor has to be compatible with aqueous solutions
- Estimated speed of the flow $v = 0.1 \dots 3 \text{ m/s}$

Basic idea: measure the pressure distribution along the surface of the

sensor and derive flow's velocity/angle of incidence



Flow characterization approach (cont'd)

Cylindrical sensor:

- Differential pressure sensors are employed
- 6 Pitot tubes distributed along cylinder's circumference
- 1 center Pitot tube as reference for the differential pressure sensors
- Estimated range of Reynolds number: $0 < Re < 15 \cdot 10^3$

Cylindrical sensor geometry:

- + Simple to fabricate
- + Well known problem in fluid dynamics
- No closed form analytical solution for pressure distribution due to **flow regimes**



Flow characterization approach (cont'd)

Analytical solution for pressure coefficient C_p around cylinder's circumference:

$$C_p = \frac{p - p_\infty}{\frac{1}{2}\rho V^2} = 2\cos(2\theta) - 1$$

- But: analytical solution only holds for potential flows (without any vorticity)
- **Thus:** equation for C_p approximates the pressure well up to the point of flow separation $(0 \le \theta \ge \frac{\pi}{2}, \frac{3\pi}{2} \ge \theta \le 2\pi)$
- For $\frac{\pi}{2} \lesssim \theta \lesssim \frac{3\pi}{2}$ different flow regimes are traversed, where the pressure significantly differs from the analytical solution above [1],[2]
- \Rightarrow A suitable data evaluation strategy has to be developed!



Flow characterization approach (cont'd)



- Constriction between sensor and side wall ⇒ a suction region on top of the cylinder
- $\rightarrow\,$ Reference pressure becomes negative
- ⇒ Positive (velocity dependent) offset!





Member of the Helmholtz Association

Cylinder with Pitot tubes





Sensor



Calibration cell



The Pressure Sensors



Member of the Helmholtz Association

Pressure Sensor Requirements

- Must be able to operate with in contact with liquids
- Pressure range depends on flow speed. Stagnation pressure:

$$p - p_{\infty} = \frac{1}{2}\rho v^2$$

 $0.1 \text{ m/s} \dots 3 \text{ m/s}$ in water $5 \text{ Pa} \dots 4500 \text{ Pa}$. \Rightarrow Rather low pressure range.

- Must be small (we need to locate many of them closely spaced)
- Not too expensive would be nice
- Easy readout is a plus.
- \Rightarrow Only a few off-the-shelf types available



Honeywell ABP2 Series

- May operate with liquid contact
- ±6895 Pa (±1 psi)



- \approx 200 Hz sampling rate
- Internal ADC, 24 bit
- SPI or I²C interface





Histograms at $\Delta p = 0$ Pa of 5 Sensors



an af the Ulaborhalter Association

Frequency Response



Member of the Helmholtz Association

Bandwidth Limitations



- Compliance of tubing, pressure sensor, trapped air
- Mass of liquid
- Friction in liquid

 \Rightarrow Spring, mass, damper system as 1^{st} order approximation.



Setup for Frequency Response Tests









Member of the Helmholtz Association

Measurement Results, misc.



Member of the Helmholtz Association

Media Compatibility Tests



Member of the Helmholtz Association

Filling of Sensors

Why?

Because a liquid/gas interface causes

- a pressure change due to the meniscus curvature
- pressure hysteresis due to contact angle hysteresis

How?

The sensor ports are very small. \Rightarrow Evacuation and filling in vacuum.



Test of Hysteresis Effects







HZDR

Member of the Helmholtz Association

Hysteresis: Unfilled sensor



Member of the Helmholtz Association

Hysteresis: Filled sensor



Long Term Test Setup

Comparison of wet and dry (expected to not get damaged) sensors with varying input pressures.





Results





Member of the Helmholtz Association

Results



Member of the Helmholtz Association

Results



Member of the Helmholtz Association

Tubing Material Tests



Member of the Helmholtz Association

Effect of Water Diffusion

- Sensor connection is like a blind hole
- Water leaves, dissolved material stays in \Rightarrow not good!





Water Diffusion

Tested Materials

■ Tygon E3603[®] (ACF00002), *d_i* = 1.6 mm, *d_o* = 3.2 mm

- VersilonTM C-210-A (AEM02002), $d_i = 1.6 \text{ mm}, d_o = 3.2 \text{ mm}$
- Tygon[®] F-4040-A (AAG00700), $d_i = 2 \text{ mm}, d_o = 3.56 \text{ mm}$
- Versilon[™] SE-200 (AJD00002), $d_i = 1.6 \text{ mm}, d_o = 3.2 \text{ mm}$
- Carl Roth Rotilabo[®] 9557.1, silicone tubing, $d_i = 1.5$ mm, $d_o = 3.5$ mm





Results Tygon E3603



DRESDEN ROSSENDOR

Member of the Helmholtz Association

Results Versilon SE-200



Member of the Helmholtz Association

Comparison



DRESDEN ROSSENDOR

Member of the Helmholtz Association

Intermediate Summary

- Filling of sensors to avoid phase boundaries is necessary.
- Expected bandwidth on the order of 100 Hz.
- No anti-aliasing filter in the ABP2 pressure sensor.
- Sensitive to trapped bubbles and setup variations.
- Sensors in solution drifted apart 200 Pa over the course of about a month.
 - Sensor damage or depositions? Unclear.
 - Short time stability seems sufficient.
- Tubing must be selected carefully
 - Tygon E3603[®] is very supple but has high water diffusion and tends to creep.
 - Versilon[™] SE-200 has low diffusion but difficult to handle.





Member of the Helmholtz Association

Measurement setup



- Prototype of the sensor used for first measurements at the calibration cell
- Only one Pitot tube (plus reference tube) is employed
- Sensor prototype can be rotated → pressure distribution around cylinder's circumference is measured for different flow rates



First measurements



- Measured at 3 different flow rates
- Mean values are shown (averaged over 10k samples)
- Agrees with theory/literature with increasing flow rates:
 - **1** Higher pressure at stagnation point $(\theta = 0)$
 - 2 In rear region, pressure decreases (flow regimes)



How to estimate the velocity/angle of incidence?

- \blacksquare Sensor consists of 6 Pitot tubes \rightarrow 6 values on the pressure distribution
- Several approaches are possible:
 - 1 Usage of a Lookup-table plus interpolation/optimizer
 - 2 Al approach
 - 3 Development of a model equation plus optimizer
- Option 1 & 2 need a lot of measurement data
- Option 3 needs less measured data and is partially physically motivated



Development of the model equation



- **1** Prior the separation point of flow, a cos-like behaviour is seen
- 2 The remaining region (rear region of the sensor) is approximated by a plateau
- **3** Weighting function is used to model the transition



Measurements vs. model equation



 Transition between cos and plateau region sufficiently good modelled

■ Currently 5 model parameters (reduction to 3 in progress) ⇒ Model resembles the shape of measured pressures



Thank you for your attention!



Member of the Helmholtz Association

Bibliography

"Frontmatter". In: Incompressible Flow. John Wiley & Sons, Ltd, 2013, pp. i-xix. ISBN: 9781118713075. DOI: https://doi.org/10.1002/9781118713075.fmatter.eprint: https://onlinelibrary.wiley.com/doi/pdf/10.1002/9781118713075.fmatter.URL: https://onlinelibrary.wiley.com/doi/abs/10.1002/9781118713075.fmatter.

William W. Willmarth and Timothy Wei. "Static Pressure Distribution on Long Cylinders as a Function of the Yaw Angle and Reynolds Number". In: *Fluids* 6.5 (2021). ISSN: 2311-5521. DOI: 10.3390/fluids6050169. URL: https://www.mdpi.com/2311-5521/6/5/169.

